## Michigan Tech CM3110 Morrison HW 5

Module	Number	Topics	Assigned Problems	Stretch Problems
5	1	oven wall heat xfer (nat'l, full and simplified)	4.7-1	
5	2	horizontal tube (nat'l)	4.7-3	
5	3	double-pipe heat exchanger	4.5-4	
5	4	change to 1-2 shell-and-tube heat exchanger	4.5-4, modified, see attached	
5	5	heat exchanger design	see attached, parts a,b	see attached, parts c-f
5	6	double pipe HE effectiveness	see attached	
5	7	heat transfer from hot pipe including radiation	4.10-3	
5	8	heat transfer from hot flat surface	see attached	
5	9	radiation shielding	4.11-1, modified	
5	10	heat exchange in top of a furnace	4.7-8	
5	11	heat exchanger choice, effectiveness	see attached	
5	12	how fouling affects U	see attached	
5	13	correlations; choose h among 3	M, attached	
5	14	natural convection h	K, attached	

## Homework 5

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- Geankoplis 4.7-1 The oven wall in Example 4.7-1 is insulated so that the surface temperature is 366.5K instead of 505.4K. Calculate the natural convection heat transfer coefficient and the heat transfer rate per meter of width. Us both the full and simplified equations. Note that radiation is being neglected in this calculation.
- 2. Geankoplis 4.7-3. A horizontal tube carrying hot water has a surface temperature of 355.4K and an outside diameter of 25.4mm. The tube is exposed to room air at 294.3K. What is the natural convection heat loss for a one meter length of pipe?
- 3. Geankoplis 4.5-4: (assume double-pipe heat exchanger; note Geankoplis' use of an improbable number of sig figs)
- 4. Geankoplis 4.5-4, except with 1-2 shell-and-tube heat exchanger:  $T'_1 = 299.5^{\circ}C$ ,  $A = 97m^2$ . How does the 1-2 shell-and-tube compare to the double pipe?
- 5. You are asked to design a counter-current double-pipe heat exchanger capable of raising the temperature of water from  $17^{\circ}C$  to  $45^{\circ}C$  (flow rate of water is 3.5 gpm). In the outside chamber of the heat exchanger will be saturated steam (pressure=  $84.55 \ kPa$ ), condensing on the outside wall of the inner pipe, through which the water flows.
  - a. How much heat (kW) needs to be removed at steady state to achieve the desired temperature change?
  - b. What is the average driving force for heat transfer for this heat exchanger?
  - c. The inner pipe is to be 2-inch schedule 40 steel pipe (inner diameter = 2.067 *in*, outer diameter  $2R_o = 2.375 in$ ). What is a good design estimate for the performance of a double-pipe heat exchanger using this pipe? I am asking you to calculate the expected overall heat transfer coefficient  $U_o$  (based on the outer diameter  $2R_o$ ) for a device with the stated pipe on the inside. You will need to construct your answer from steam-side  $(h_2)$  and water-side  $(h_1)$  heat transfer coefficients obtained from data correlations associated with the appropriate heat-transfer physics (forced convection? natural convection? radiation? condensing vapor? other?). We desire the length of the heat exchanger to be no more than 1.8 *m*. (*HINT: The steam-side heat transfer coefficient is most easily found using a correlation in Perry's (section on heat transfer with change of phase; Perry's section on thermal design of heat transfer equipment has some typical values of U to check your work.))*
- 6. Water flowing at a rate of 0.723 kg/s enters the inside of a countercurrent, doublepipe heat exchanger at 300. K and is heated by an oil stream that enters at 385K at a rate of 3.2 kg/s. The heat capacity of the oil is 1.89kJ/kg K and the average heat capacity of water over the temperature range of interest is 4.192 kJ/kgK. The overall heat-transfer coefficient of the exchanger is  $3.0 \times 10^2 W/m^2K$ , and the area for heat transfer is  $15.4 m^2$ . What is the total amount of heat transferred? Answer: 175kW.
- 7. Geankoplis 4.10-3. A horizontal oxidized steel pipe carrying steam and having an OD of 0.1683m has a surface temperature of 374.9K and is exposed to air at 297.1K in a

large enclosure. Calculate the heat loss for 0.305m of pipe from natural convection plus radiation. For the steel pipe use an emissivity of 0.70.

- 8. A horizontal heated plate of dimensions 1.0 by 1.0 m is heated to  $50^{\circ}C$ . The air over the plate is at a temperature of  $25.0^{\circ}$ C. What is the total heat flux from the plate to a body that is the same temperature as the air? The emissivity of the plate is 0.89. What mechanism dominates heat transfer (if any)?
- 9. Geankoplis 4.11-1, section c) added. Two very large and parallel plates each have an emissivity of 0.7. Surface 1 is at 866.5K and surface 2 is at 588.8K.
  - a. What is the net radiation loss of surface 1?
  - b. To reduce this loss, two additional radiation shields also having an emissivity of 0.7 are placed in between the original surfaces. What is the new radiation loss?
  - c. How many radiation shields would be needed to drop the radiation loss to  $3500 W/m^2$ ? Answers:

a. 
$$40,000 \frac{W}{m^2}$$

b. 
$$4500 \frac{m}{m}$$

b.  $4500 \frac{1}{m^2}$ c. Add 1 more radiation shield

- 10. Geankoplis 4.7-8: Two horizontal metal plates having dimensions of  $0.8 \times 1.0m$ comprise the top of a furnace and are separated by a distance of 15mm. The lower plate is at 400.° C and the upper at 100.° C, and the air at atmospheric pressure is enclosed in the gap. Calculate the heat transfer rate between the plates. We need to calculate radiation and natural convection contributions to the total. Answers: radiation contribution 5.5kW; natural convection contribution 1.3 kW; total 6.8 kW.
- 11. The liquid feed stream to a reactor needs to be preheated from  $25^{\circ}C$  to at least  $63^{\circ}C$ . The stream has a mean heat capacity of 6.5  $kJ/kg^{o}C$  and the mass flow rate is 2.44 kg/s. There is a hot stream from a nearby process available to use in the outside of a double-pipe heat exchanger (counter current flow) to bring about the heating. The hot stream fluid is mostly water (average density =  $997kg/m^3$ ; mean heat capacity = 4.2kJ/kgK; average viscosity= $9.2 \times 10^{-4}kg/ms$ ), and this stream is available at approximately  $90^{\circ}C$ , flowing at 45 gpm (gallons per minute). Three heat exchangers are available from your company's usual vendor (see table below). Make the appropriate calculations to determine which of the three (if any; there also may be more than one) would be appropriate for the job. Which heat exchanger(s) do you recommend? You must justify your selection with calculations.

Heat	u(kW)	$A(m^2)$	Flow
exchanger	$O\left(\frac{1}{m^2K}\right)$		direction
А	300	0.080	counterflow
В	330	0.090	counterflow
С	500	0.095	counterflow

12. Fouling is a serious concern with heat exchangers. We wish to evaluate the magnitude of the likely fouling effect on a double-pipe heat exchanger with city water flowing on the outside and orange juice (a vaporizing liquid) on the inside.

- a. For a double-pipe heat exchanger (inner pipe made of nominal 2-inch schedule 40 steel pipe) and assuming a logarithmic middle value for the likely values of heat transfer coefficients on the inside  $h_i$  and outside  $h_o$  (see Geankoplis Table 4.102, given in the answers to HW 5; "logarithmic middle" means half-way between values on a log scale), what is your estimate of the overall heat transfer coefficient? How does your value compare with typical values for U? (see Geankoplis Table 4.9-2). Base your overall heat transfer coefficient on the outside radius of the inner pipe,  $U_o$ .
- b. After some amount of time in service, we expect that heat exchanger fouling will affect the amount of heat transferred in the heat exchanger. What is your estimate of the expected change in U due to the city water fouling on the outside of the heat exchanger? What is your estimate of the expected change in U due to the orange juice fouling (vaporizing liquid) on the inside of the heat exchanger? What would be the combined effect? Comment on your answers.
- Between these three correlations for heat transfer coefficient h, which would you use in the circumstance described below? Why and with what conditions? Is radiation important? Calculate the total heat loss including radiation if necessary.

a. 
$$Nu_a = \frac{h_a D}{k} = 1.86 \left( RePr \frac{D}{L} \right)^{\frac{1}{3}} \left( \frac{\mu_b}{\mu_w} \right)^{0.14}$$

b. Nu = 
$$\frac{hL}{k} = aGr^m Pr^{\frac{2}{3}}$$

c. Nu = 
$$\frac{hL}{k} = cRe^m Pr^{\frac{1}{3}}$$

A horizontal pipe (Schedule 40, outer diameter 2.375 in; inner diameter 2.067 in; steel) connects two tanks in a pilot plant. The hot oil flowing in the tube heats the pipe to an outside surface temperature of  $116^{\circ}C$ . A fan blows across the pipe, sending a steady flow of  $15^{\circ}C$  air (1.0 atm) across the tube at 12.0 m/s. What is the heat loss  $(kW/m^2)$  from the pipe?

14. Problem K: A wide, deep rectangular oven (1.0 ft tall) is used for baking loaves of bread. During the baking process the temperature of the air in the oven reaches a stable value of  $100^{\circ}F$ . The oven side-wall temperature is measured at this time to be a stable  $450^{\circ}F$ . Please estimate the *natural convection* heat flux from the wall per unit width. (The other contribution will be *radiation*, which is important.)