

CM 3230
Fall 2016
Exam 1

Name _____

1. (20 pts) A rigid vessel contains two compartments of the same volume separated by a thin membrane and is in equilibrium with the surrounding temperature, $T_{init} = T_{surr} = 25^\circ\text{C}$. One compartment contains 10 moles of ideal gas A, having a heat capacity $c_{P,A} = 3.2R$, while the other compartment contains 15 moles of ideal gas B, having a heat capacity $c_{P,B} = 1.5R$. Afterwards, the membrane ruptures and both gases mix completely, settling to one pressure and to the same surrounding temperature, $T_{final} = T_{surr} = 25^\circ\text{C}$. Calculate the total change in entropy of the universe (in kJ/K) from the point before rupture to the point where it equilibrates to the final pressure and temperature.
2. (40 pts) An engine containing an ideal gas having molar heat capacity $c_p = 3R/2$ operates with a cycle shown in Figure 1. At point a , the gas expands isobarically at P_a to point b . Then it expands adiabatically from point b to point c where the pressure is at point c is given by $P_c = 2P_b/3$. Afterwards, it is compressed isobarically to point d . Finally, it is compressed adiabatically back to point a . All the paths are assumed to be reversible. The temperatures at points a and c were measured to be $T_a = 110^\circ\text{C}$ and $T_c = 250^\circ\text{C}$. Calculate the net work done by each mole of the gas (in kJ/mol) for one cycle and the molar heat input to the gas (in kJ/mol) as it expands from point a to point c .

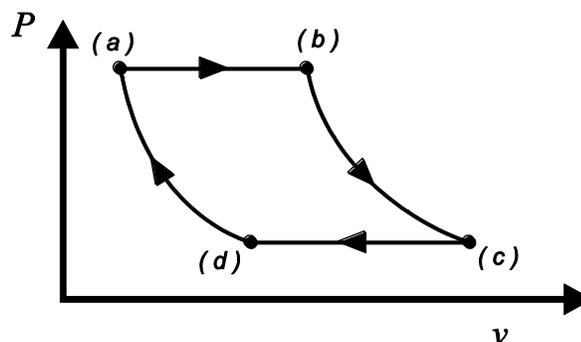


Figure 1. Engine cycle with paths $a \rightarrow b$ and $c \rightarrow d$ reversible isobaric, while paths $b \rightarrow c$ and $d \rightarrow a$ are reversible adiabatic.

(Note: More questions on the back page.)

3. (20 pts) A refrigerant with quality $x_{in} = 0.2$ flows adiabatically through the throttle from $P_{in} = 3 \text{ bars}$ and $T_{in} = 80^\circ\text{C}$ to $P_{out} = 2.5 \text{ bars}$ and $T_{out} = 60^\circ\text{C}$. Calculate the specific entropy change (in $\text{kJ}/(\text{kg} \cdot \text{K})$) of the refrigerant as it undergoes the throttling process using the following data (based on a common reference condition):

Data: 1) @ (T_{in}, P_{in}) :

$$\hat{h}_{in,vap} = 2824 \frac{\text{kJ}}{\text{kg}} \quad ; \quad \hat{s}_{in,vap} = 7.52 \frac{\text{kJ}}{\text{kg} \cdot \text{K}}$$

$$\hat{h}_{in,liq} = 450.7 \frac{\text{kJ}}{\text{kg}} \quad ; \quad \hat{s}_{in,liq} = 1.44 \frac{\text{kJ}}{\text{kg} \cdot \text{K}}$$

2) @ (T_{out}, P_{out}) :

$$\hat{h}_{out,vap} = 2550 \frac{\text{kJ}}{\text{kg}} \quad ; \quad \hat{s}_{out,vap} = 8.31 \frac{\text{kJ}}{\text{kg} \cdot \text{K}}$$

$$\hat{h}_{out,liq} = 90.3 \frac{\text{kJ}}{\text{kg}} \quad ; \quad \hat{s}_{out,liq} = 0.687 \frac{\text{kJ}}{\text{kg} \cdot \text{K}}$$

4. An ideal gas of $c_p = 3R/2$ inside a piston-cylinder undergoes a sudden pressure drop from initial pressure P_{init} to a constant external pressure P_{ext} causing an irreversible expansion of the gas. The final pressure is $P_{final} = (0.4 P_{init}) = P_{ext}$ and $T_{final} = T_{surr}$. The expansion ratio was also found to be $\hat{v}_{final}/\hat{v}_{init} = 3$.
- (20 pts) Evaluate the molar change in entropy of the gas during the expansion process in $\text{J}/(\text{mol} \cdot \text{K})$.
 - (Bonus: 5 pts) Will $T_{final} > T_{init}$ or $T_{init} > T_{final}$? Explain using equations.